

Numerical Studies on Heat Transfer Behavior of Electronics Module with Zr, SiC and Cu Water Based Nanofluids

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ABSTRACT

Normal air cooling method is not pertinent for high heat flux electronics apparatuses. For that, the thermal investigation of electronics module is really indispensable towards its hassle free operation. The present study involves an electronics module kept horizontally at the base, inside a square shaped chamber filled with nanofluid as coolant. Three different water based nanofluids, specifically Water-Zr, Water-SiC and Water-Cu, are considered as coolants in the present investigations. The numerical studies are performed to obtain the heat transfer behavior of electronics module for maintaining its temperature within the safe limit. For that, a 2D numerical model is being developed which also includes thermal buoyancy. The continuity, momentum and energy equations are solved to predict the thermal behavior. Simulations are conducted to predict the temperature fields and temperature contours. The trends of results are along the expected lines. Simulation results predicted with three different water based nanofluids are analyzed and compared for realizing the relative importance of the stated nanofluids. The key model parameter considered is heat flux of 70 W/cm² associated with the electronics module. The Water-SiC is witnessed as the nanofluid delivering comparatively greater cooling behavior towards electronics module with no such failure as a result of heat.

Keywords: Electronics Module, Simulation, Nanofluids, Water-Zr, Water-SiC, Water-Cu.

I. INTRODUCTION

Modern fashion towards miniaturization of electronic packages accompanying the limitlessly more circuit densities has caused enormously high power densities. This tendency towards compactness involves high heat flux in various applications and has provided motivation, during the past several years, for significant volume of research related to the design and development of effective cooling schemes. In view of the present trend of continual increase in both packaging and power densities in modern day's electronics gadgets, the search for the suitable cooling techniques, depending on the applications, motivated the investigators all over the world. As the simple free or forced convection air cooling exercise is unsatisfactory for the high heat flux usages, the race for several ways and means of cooling have congregated much prominence in recent years to surpass the trials of enormous heat related to the electronic objects.

Wadsworth and Mudawar [1] reported on heat transfer behavior of a multichip electronic module with confined 2D jets of dielectric liquid. Webb and Ma [2] studied about single phase liquid jet impingement heat transfer. Xuan and Roetzel [3] discussed about the conceptions of heat transfer correlation of nanofluids. Basak et al. [4] reported on effects of thermal boundary conditions on natural convection flows within a square cavity. He et al. [5] described about heat transfer and flow behaviour of aqueous suspensions of TiO2 nanofluids flowing upward through a vertical pipe. Anandan and Ramalingam [6] reviewed on thermal management of electronics. Kurnia et al. [7] analyzed numerically on laminar heat transfer performance of various cooling channel designs. Yang and Wang [8] simulated a 3D transient cooling portable electronic device using phase change material. Zhu et al. [9] optimized the heat exchanger size of a thermoelectric cooler used for electronic cooling applications. Gong et al. [10] presented numerically on layout of microchannel heat sink useful for thermal management of electronic devices. Naphon et al. [11] considered thermal progress practices for electronic objects.

From the referred revisions, to the best of author' awareness, it is understood that there is not a single complete computational study connecting to the influences of water based nanofluids (actually Water-Zr, Water-SiC and Water-Cu) on heat related challenges of electronics modules. With this perspective, the present paper demonstrates numerical investigations with the stated nanofluids on thermal characteristics of electronics modules. And also, the numerical model includes additional key factors like inertia, viscosity and gravity effects apart from the usual issues concerning the present physical problem. However, the stated model ignores both compressibility and viscous heat dissipation effects. The model is very well demonstrated for the detailed numerical investigations on the influences of the already stated nanofluids (as they significantly affect the cooling characteristics) by taking electronics module heat flux and duct inlet nanofluid velocity as the important model parameters. In due course, the model predictions pertaining to the enumerated nanofluids are along the expected lines as well.

II. DESCRIPTION OF PHYSICAL PROBLEM

The neat sketch of a common electronics module representing the base of a square shaped chamber is very well described in the figure 1. It illustrates on the overall heat transfer from the electronics module kept horizontally at the base of square shaped chamber. The coolants considered in the present investigations are three different water based nanofluids named as Water-Zr, Water-SiC and Water-Cu. A 2D model is considered to save computation/simulation time by ignoring end effects in the transverse direction. The model includes thermal buoyancy, viscosity along with the gravity effect as well. The fluid flow is considered to be laminar and incompressible. The ambient together with the no slip boundary condition is specified at the walls. For cooling of the electronics module, a convective boundary condition in the form of heat flux is introduced at the base to simulate the overall temperature variation inside the square chamber due to heat transfer. The thermophysical properties of listed nanoparticles in combination with the additional model data, are mentioned in table 1.



Figure 1. Schematic illustration of electronics module computational domain

NanoparticleProperties	Zr	SiC	Cu
Density, ρ (Kg/m ³)	5680	3160	8940
Specific heat, C_P (J/kg-K)	418	675	385
Thermal conductivity, k	22.6	490	401
(W/m-K)			
Model Data	Values		
Height/Width of chamber	60 mm		
Electronics module length	60 mm		
Ambient air temperature	300 K		
Electronics module heat flux	70 W/cm^2		

III. MATHEMATICAL FORMULATION

The related physical problem is converted into a set of governing transport equations which are solved with the accompanying computational trials on the subject of both modeling and simulation. The related continuity, momentum and energy equations in 2D for a fully developed hydrodynamic and thermal flow situations are described in equations from (1) to (4), respectively. The compressibility and the viscous heat dissipation effects are ignored in the present physical situation. However, the thermal buoyancy factor (represented by $\rho g\beta \Delta T$) is incorporated in y-momentum equation (3).

Continuity equation:
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$
 (1)

X-momentum equation:

$$\rho\left(\frac{\partial u}{\partial t} + u\frac{\partial u}{\partial x} + v\frac{\partial v}{\partial y}\right) = -\frac{\partial P}{\partial x} + \mu\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right)$$
(2)

Y-momentum equation:

$$\rho\left(\frac{\partial v}{\partial t} + u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y}\right) = -\frac{\partial P}{\partial y} + \mu\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) + \rho g\beta\Delta T$$
(3)

Energy equation:

$$\left(\frac{\partial T}{\partial t} + u\frac{\partial T}{\partial x} + v\frac{\partial T}{\partial y}\right) = \alpha \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}\right) \tag{4}$$

IV. NUMERICAL PROCEDURES

A. Numerical Scheme and Solution Algorithm

The related governing transport equations are modified into generalized form as follows.

$$\frac{\partial}{\partial t}(\rho \varphi) + \nabla .(\rho \mathbf{u} \varphi) = \nabla .(\Gamma \nabla u) + S$$
(5)

The modified governing transport equations are discretized with the second order upwind scheme by means of a pressure based finite volume technique with the SIMPLER algorithm, where Γ represents a transport property (*k* or μ), ϕ specifies any conserved parameter and *S* is a source term.

In the beginning, both the continuity and momentum equations are solved at a time for generating the pressure and velocity fields. Subsequently, the energy equation is solved by expending the identified velocity field to get the related temperature field. In other words, all the associated equations are solved together (but not independently) bearing in mind the interdependency between the associated variables.

B. Choice of Grid Size, Time Step and Convergence Criteria

A general grid-independence test is executed to establish the correct spatial discretization, and the levels of iteration convergence criteria to be worthwhile. As an outcome of this test, we have used 60×60 uniform grids for the final simulation. Corresponding time step taken in the simulation is 0.0001 seconds. Though we checked with smaller grids of 90 and 120 in numbers for 60 mm width/height of the computational domain, it is observed that a finer grid system does not alter the results significantly. In other words, the statistical data reveals that the finer grids have minor effect in the simulation results which is quite obvious from the definition of grid-independence test. Additionally, the smaller grid requires more computational time vis-à-vis more constancy in predictions of several fields/contours.

Convergence in inner iterations is assured only when the condition $\left|\frac{\varphi-\varphi_{old}}{\varphi_{max}}\right| \leq 10^{-4}$ holds good for all variables simultaneously, where φ stands for each variable u, v, and T at a grid point at the current iteration level, φ_{old} represents the corresponding value at the previous iteration level, and φ_{max} is the maximum value of the variable at the current iteration level in the entire domain.

V. RESULTS AND DISCUSSION

Numerical simulations are executed to examine the effects of three different water based nanofluids (i.e. Water-Zr, Water-SiC and Water-Cu) on heat transfer characteristics of electronics module in terms of temperature distributions (i.e. temperature contours/fields) and surface temperatures of electronics modules. To begin with, the size of the square chamber is selected as 60 mm. Besides, the electronics module is subjected to a heat flux of 70 W/cm².

A. Case study with Water-Zr nanofluid as coolant

With the enumerated model conditions, with the purpose of studying the effect of Water-Zr nanofluid on the thermal behavior of the electronics module, the numerical simulations are performed, by introducing the thermophysical properties of the stated nanofluid into the model.

Figure 2 illustrates the simulated results of the temperature field (alongside the colored scale bar unveiling the temperature values in terms of K) as observed at the stated model conditions by considering the Water-Zr nanofluid as coolant. The surface temperature of electronics module is found to be 340 K (which is relatively nearer to the safe limit of 356 K temperature as desired in order to avoid the thermal failure of the electronics module). As expected, the temperature of the Water-Zr nanofluid is maximum near the vicinity of electronics module. And also, the temperature of the Water-Zr nanofluid gradually decreases with the increase in the distance from the

electronics module and then it becomes equal to the atmospheric temperature in the far field region. The associated temperature contour is also shown in figure 3. Furthermore, the trends of results are also along the expected lines.



Figure 2. Temperature field with Water-Zr nanofluid as coolant



Figure 3. Temperature contour with Water-Zr nanofluid as coolant

B. Case study with Water-SiC nanofluid as coolant

With the stated model conditions, with the intention of studying the effect of Water-SiC nanofluid on the thermal behavior of the electronics module, the numerical simulations are performed, by introducing the thermophysical properties of the enumerated nanofluid into the model.

Figure 4 demonstrates the simulated results of the temperature field (along with the colored scale bar displaying the temperature values in terms of K) as obtained at the stated model conditions by considering Water-SiC nanofluid as coolant. The surface temperature of electronics module is found to be 309 K (which is far below the safe limit of 356 K temperature as desired in order to avoid the thermal failure of the electronics module). As expected, the temperature of the Water-SiC nanofluid is maximum near the vicinity of electronics module. And also, the temperature of the Water-SiC nanofluid gradually decreases with the increase in the distance from the electronics module and then it becomes equal to the atmospheric temperature in the far field region. The related temperature contour is also illustrated in figure 5. The trends of results are along the expected lines as well.







Figure 5. Temperature contour with Water-SiC nanofluid as coolant

C. Case study with Water-Cu nanofluid as coolant

With the enumerated model conditions, with the purpose of studying the effect of Water-Cu nanofluid on the thermal behavior of the electronics module, the numerical simulations are conducted, by incorporating the thermophysical properties of the specified nanofluid into the model.

Figure 6 illuminates the simulated results of the temperature field (combined with the colored scale bar unveiling the temperature values in terms of K) as observed at the specified model conditions by considering Water-Cu nanofluid as coolant. The surface temperature of electronics module is found to be 319 K (which is also within the safe limit of 356 K temperature as desired in order to avoid the thermal failure of the electronics module). As expected, the temperature of the Water-Cu nanofluid is maximum near the vicinity of electronics module. And also, the temperature of the Water-Cu nanofluid gradually decreases with the increase in the distance from the electronics module and then it becomes equal to the atmospheric temperature in the far field region. The corresponding temperature contour is also illustrated in figure 7. The trends of results are also along the lines of expectations.



Figure 6. Temperature field with Water-Cu nanofluid as coolant



Figure 7. Temperature contour with Water-Cu nanofluid as coolant

D. Comparison of predicted temperatures of electronics modules obtained with different nanofluids as coolants

Table 2 recapitulates the numerically predicted temperatures of the electronics modules as observed by

using three different water based nanofluids (specifically, Water-Zr, Water-SiC and Water-Cu) as coolants. It is noticed that the numerical predictions/results are comparable with each other. As expected, the variations in the numerically predicted temperatures of the electronics modules are witnessed very clearly with the use of the stated water based nanofluids as coolants. This is as a result of the variations in the thermal conductivities of the associated nanoparticles as mentioned in table 1.

Table 2. Comparison of numerical predictions ofelectronics modules temperatures with differentnanofluids as coolants

Name of	Numerically Predicted Temperature	
Nanofluid	of Electronics Module (K)	
Water-Zr	340	
Water-SiC	309	
Water-Cu	319	

Similarly, figure 8 also demonstrates the plot representing the variations in the electronics modules temperatures with three different water based listed nanofluids as coolants. It is seeming that the trends of the variations in the numerically predicted results are along the lines of expectations.



Figure 8. Variations in electronics modules temperatures with different water based nanofluids as coolants

VI. CONCLUSION

A 2D numerical model pertaining to the electronics module is developed to predict the heat transfer issues

using three different water based nanofluids, namely Water-Zr, Water-SiC and Water-Cu as coolants. The model includes additional key factors like inertia, viscosity, gravity and thermal buoyancy effects apart from the usual issues concerning the present physical problem. However, the specified model ignores both compressibility and viscous heat dissipation effects. The model is very well demonstrated for the detailed numerical investigations on the influences of the already stated nanofluids (as they significantly affect the cooling characteristics) by taking electronics module heat flux of 70 W/cm^2 as the important model parameter. The predictions of the model pertaining to the different nanofluids are along the expected lines. Direct comparison with other numerical models of electronics modules is not possible because of the absence of such models in the literature. However, the experimental comparison with an in-house experimental setup is planned for the future. With the enumerated model conditions, it is observed that the Water-SiC nanofluid delivers appropriately suitable cooling behavior without any kind of thermal failure and is the superior one as the electronics module temperature is far below the safe limit. Consequently, the identified model combined with the nanofluid can be utilized right away in industries for electronics modules cooling and to enhance heat transfer.

VII. REFERENCES

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